

ENERGY STAR Multifamily New Construction, Version 1 / 1.1 / 1.2 (Rev. 032)

HVAC Designer Responsibilities:

- Complete one National HVAC Design Report for each building which includes system design for all unique unit plans and common spaces. For
 projects with multiple buildings, one National HVAC Design Report per building or per project is permitted. ¹
- Obtain efficiency features (e.g., window performance, insulation levels, and infiltration rate) from the builder, architect, or Rater.²
- Provide the completed National HVAC Design Report to the Rater and the person / company completing the National HVAC Functional Testing Checklist.²

1. Design Overview							
1.1 Designer name:	Designe	r company:			Date: _		
1.2 Select which party you are providing these design serv							tor
1.3 Name of company you are providing these design serv	ices to (if differe	ent than Item 1	.1):				
Name of company you are providing these design serv A ProjectBuilding address:		City:		Sta	ite: Zip	code:	
2a. Dwelling Unit & Common Space Mechanical Ven	itilation Desig	n ("Vent Sys	stem") ³ & Inle	ets in Return [Ouct ^{4, 5, 6}	Desig Verif	
Airflow:							
2.1 Dwelling unit ventilation airflow design rate & run-time reari = 2010 = 2013.	meet the require	ements of Sec	tion 4 of ASHR	AE 62.2 ⁷ <u>– 20</u>	<u>[enter</u>]
2.2 Common space outdoor airflow design rate meet the re	•				<u>/ear]</u> ⊟-2010-⊟]
ERI and Prescriptive Path Only: Rates shall not exceed							
2.3 Access points to measure airflow rate and inspect outd	oor air dampers	are provided	and accessible	by the Rater. ²	. <u>. 9</u>		l
List unique unit plan for which 62.2 ventilation rates were calculated in the spaces to the right: 9							
2.4 # of bedrooms:							
2.5 Square footage:							
2.6 Ventilation airflow rate required by ASHRAE 62.2:							
2.7 Ventilation airflow rate designed:							
2.7.1 If applicable, run-time per cycle (minutes):							
2.7.2 If applicable, cycle time (minutes):							
List common space for which 62.1 ventilation rates were calculated in the spaces to the right: 9,810,11							
2.8 Ventilation airflow rate-required by ASHRAE 62.1 (CFM):							
2.9 Ventilation airflow rate designed (CFM):							
System Type & Controls:							
List Ventilation System ID in the spaces to the right: 109							
2.10 Specified system type: (e.g., supply, exhaust, balanced, ERV, HRV)							
2.11 Specified system type: (e.g., in-unit, central)							
2.112 Manufacturer:							
2.123 Model Number:							
2.13 # installed in the building:							
2.14 Spaces each fan serves (i.e., single, multiple)							
2.1 <u>5</u> 4 Area / space(s) that system serves: (e.g., Unit A kitchens, corridor, community room)							
2.165 Specified control location: (e.g., Master bath, utility):							
2.1 <u>76</u> Specified controls allow the systems to operate auto							
specified and also labeled if its function is not obvious (e.g. the ventilation equipment). In townhouses only, this control dwelling units, the override control is not required to be rearrecommends but does not require that the control be readil occupant.	must be readily dily accessible	/ accessible to to the occupar	the occupant. nt. However, in	In all other mul such cases, EF	tifamily PA]
2.47_18 For any outdoor air inlet designed to connect to the automatically restrict airflow using a motorized damper dur					controls		□ N/A
Sound:	3 73.7	5,515 4114 0	22494111 0 101110				<u>,, ,</u>
2.198 If located in the dwelling unit, the fan of the specified exempted. 1344	d system is rate	d ≤ 3 sones if	intermittent and	d ≤ 2 sones if co	ontinuous, or		□ N/A
Efficiency:							<u>. 47.4</u>



2.2019 If dwe either the fan cooling hours	type in Iter															□ <u>N/A</u>
2.2 <mark>1</mark> 0 If in-uni				e specified a	s part of	f the <u>D</u> dwell	ling <u>L</u>	<u>J</u> unit <u>M</u> mo	echa	nical <u>\</u>	<u>∕</u> ventilatio	on <u>S</u> syste	m, the	en		□ N/A
2.224 If central direct-drive, E	al exhaust CM, with v	fans, ≤ 1 ŀ	HP, are spec													□ <u>N/A</u>
Air Inlet Loca		mplete th	is section if s	ystem has s	pecified	air inlet loc	ation	(s); other	rwise	check	"N/A".)	<u>16</u> 14			Ve	signer erified N/A
2.2 <mark>32</mark> Inlet(s)	pull ventila	tion air di	rectly from ou	ıtdoors and i	not from	attic crawl	Ispac	e garage	e or	adiace	nt dwelli	na unit				
2.2 <u>43</u> Inlet(s) vent, exhaust	are ≥ 2 ft. a	above gra	de or roof de	ck; ≥ 10 ft. o	f stretch	ned-string di	istand	ce from k	now	n conta	amination		(e.g.,	, stack,		
2.254 Inlet(s)	•						<u>u 500</u>	11000 0/11	iiig t	110 100						
2b. Dwelling dwelling unit l	Unit Loc	al Mecha	ınical Exhai	ust Design	- Syste	m(s) are de								ates.		
Location		Continu	ous Rate			Intermitte	nt R	ate ¹⁸¹⁶					Exh	aust F	an T	vpe
Kitchen	Airflow		, based on ki	tchen volum		≥ 100 CFM ACH based	1 and	, if not int	tegra	ited wit	h range,	also ≥ 5 220	□С	ontinuo	us	7 -
	Sound	Recomm	ended if in-u	nit: ≤ 1 sone		Recommer							□ In	n-unit fai entral /	n	ad fan
Bathroom	Airflow	≥ 20 CFN	Л			≥ 50 CFM							_	ontinuo		Ju Turi
	Sound		I if in-unit: ≤ 2	2 sones		Recommended if in-unit: ≤ 3 sones ☐ Intermitte						tent an				
2c. Commor						System(s) a	are d	esigned t	that i	mecha	nically ex	khaust air		entral /	share	ed fan
common space	ce, as requ	_											_			
Location		ł	E 62.1 Rate	Design Ra	te	Location		1.96.1	2221		RAE 62.		Des	ign Ra	te	
Janitor Room		1 cfm/ft ²				Common s	•			50 cm	n / 100 c	rm				
Trash / Recyc	ling Room		/m2 - 1 II			Common s	расе	Datillooi	111	50 cfn	n per toil	et / urinal				
Parking Gara	ge		/ft ² , standby /ft ² , full-on			☐ Shared (garag	ge exhau	st fa	n contr	ols inclu	de CO an	d NO	2 senso	rs.	
3. Heating &																
3.1 Loads cal per AHJ ²⁶²⁴ Townhous	culated usi	ng: 🗆 Una		CA Manual J	v8 _⊏	•										□ N/A □ Other
3.2 Check on ☐ Unit-specifi☐ Worst-case units if cooling	ic design e design (If g system se	the top flo elected for	☐ Group desi oor unit with t r all is single-	gn ²⁸²⁶ he greatest speed & <20	total gro CFA and kBtuh	oups for this d window ar or two-spee	s proj rea re ed / va	ect<u>buildir</u> esults in t ariable-sp	<u>ng</u> , re otal	eprese heat ga	nting ain <18 k	units.				
3.3 Indoor de																
3.4 Outdoor d	•			: (See Footr	ote <u>29</u> 2	4 and www	.ener									
			y selected: _	100				Coo	oling	seaso	า:	_°F	leatin	g seaso	n:	°F
List the unit	-															
3.5 Location of 3.6 Number of							-									
3.7 Total occu																
3.8 Condition		, ,		29											-	
3.9 Window a																
3.10 Predomi				· <u>2725, 33</u> 31												
3.11 Infiltratio																
3.12 Mechani	•						\dashv						-		+	
3.13 Non-occ		nal gains	(appliance, e													
3.14 Door o		<u> </u>		W. NW): 2826			+					1			+	
3.15 Sensible							\dashv								\vdash	



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3.16 Latent Heat Gain At Design Condition	ons (kBtuh):									
3.17 Total Heat Gain at Design Condition	ns (kBtuh): 2725									
3.18 Total Heat Loss at Design Condition	, ,							_		
	()		L						Desig	aner
3.19 Common Space Heating & Coo	ling Loads ¹⁰⁹ (re	quired for	all common	space hea	ting and co	oling sys	stems)		Veri	
									1	√ <u>/A</u>
Common Space Name:	Design Condition	ons: Total I	Heat Gain: _	(kBt	uh) To	tal Heat l	_oss:	(kBtul	h)	
Common Space Name:	Design Condition	ons: Total I	Heat Gain: _	(kBt	uh) To	tal Heat l	_oss:	(kBtul	h)	
Common Space Name:	Design Condition	ons: Total I	Heat Gain: _	(kBt	uh) To	tal Heat l	_oss:	(kBtul	h)	
3.20 Building Heating & Cooling Loa	ids 109 (only requi	red when s	shared syste	ms such a	s central b	oilers or	chillers a	are specifie	ed.)	N/A
Designer	` , , ,		·					·	-	
Verified										
E-N/A	D i O diti	T-4-1 I	In at Online	/I-D4	L.\ T			(I-D4-I		
System Name:	Design Conditi			(kBt		al Heat L		(kBtuh	,	
System Name:	Design Conditi	ons: rotai i	neat Gain: _	(kBt	un) ro	al Heat L	.088:	(kBtuh	1)	
4. Heating & Cooling Equipment Sel		-4!!	I- 045		(0 [2520\ 33			
4.1 Equipment selected per ☐ ACCA Ma					`	ootnote		5 1: (1		
4.2 Prescriptive Path: Equipment serving Exhibit X of the National Rater Field Che	dwelling units <u>.</u> and cklist. Electric resis	common stance space	spaces <u>, and</u> ce heating is	garages me not specifie	eet the effic ed in dwellin	iency levi g units. 3	els specit 6 -	ied in the		□ N/A
4.3 ERI Path: Equipment serving commo the Exhibit X of the National Rater Field (cified in		□ N/A
Cooling Equipment 9 (Complete all ap							_	d in multin	le sna	
(columns), identical data is not require										N/A
List Cooling Equipment ID in the spaces duplicating as needed for each unique										
4.4 Equipment type: (e.g., PTAC / AC, Cl WLHP / GSHP / ASHP / VRF)	hiller / CT, PTHP /									
4.5 Area / Space(s) that system serves:										
4.6 Chiller / condenser / outdoor unit mar	nufacturer:			1						
4.7 Chiller / condenser / outdoor unit mod	del #·			+						
4.8 Evaporator / indoor unit manufacture				+						
4.9 Evaporator / indoor unit model #:	•			+						
4.10 AHRI reference #: 3734				+						
4.11 AHRI-Listed efficiency:				+						
4.12 Evaporator fan type: PSC, ECM / IC	M Other			+						
4.13 Compressor speed: Single, Two, Va	•			+						
4.14 Turn down ratio (for variable speed				+						
4.15 Latent capacity at design conditions	· · · · · · · · · · · · · · · · · · ·			+						
4.16 Sensible capacity at design condition	, ,			+						
4.17 Total capacity at design conditions (+						
4.18 Cooling sizing % = Total capacity (It	·			+						
by Total Heat Gain (Item 3.17) of space(s										
4.19 Meets cooling sizing limit: (see below N/A) 2523, 27	w for A, B, C, D or									
4.20 If "B", list Load sensible heat ratio = heat gain (Item 3.15) / Max. total heat ga										
4.21 If "B", calculate HDD / CDD ratio: 395				+					 	
4.21 II B , Calculate 11DD / CDD Tatio. —			Comp	TOO OF THE	o (Dor Itom	. 4 42\				
Fautinment Type & Climate Condition	Cinalo Ci		Comp	ressor Typ	•	14.13)	Voris	abla Casad		
Equipment Type & Climate Condition A: For Cooling-Only Equipment or	Single-S	peeu		Two-Spee	u		varia	able-Speed		
For Cooling Mode of Heat Pump in	Recommended Allowed: 90			mmended: 90 -				ended: 90 - ed: 90 – 16		Ď
Condition A Climate 3936 B: For Cooling Mode of Heat Pump in Condition B Climate 3936	90% - 100%, p	lus 15 kBtu	ıh 90%	- 100%, plu	s 15 kBtuh		90% - 10	0%, plus 1	5 kBtuł	า
C: For low-load spaces (≤15 kBtuh) 4037	≤ 20 kl									
D: For low load spaces (<18 kBtuh) 4037	= 20 Ki			< 25 kBt	uh			25 kBtub		



ENERGY STAR Multilatility	y inew C	วเารแนบ	uon, ve	151011 1 /	1.1/1.2	(Nev. U	<u>3≠</u>)
Heating Equipment ⁹ (Complete all applicable items, no multiple spaces (columns), identical data is not required							Designer Verified
check "N/A".)							□ N/A
List Heating Equipment ID in the spaces to the right; duplicating as needed for each unique space served:							
4.22 Electric equipment type: PTHP, WLHP, GSHP, ASHP, VRF, Boiler, Furnace, Electric Resistance							
4.23 Gas Equipment type: HW PTAC / fan coil, Gas-Fired PTAC, Boiler, Furnace							
4.24 Area / Space(s) that system serves:							
4.25 Manufacturer:							
4.26 Model Number:							
4.27 AHRI reference #: 37							
4. 27- 28 Listed efficiency:							
4.298 Equipment output capacity (kBtuh): 41							
4.3029 Air-source heat pump output capacity (17°F)							
4.310 Type of Venting: Natural Draft, Mechanically Drafted, Direct Vent 4238							
4.324 Furnace heating sizing % = Total capacity (Item 4.289) divided by Total Heat Loss of space(s) in Item 4.24:							
4.332 Meets furnace sizing limit: (see below for A, B, C, or N/A) 2523							
A: For low-load spaces	(≤ 10 kBtuh)	, furnace o	utput capaci	ty is ≤ 40 kB	tuh	I	I
B: When Used for Heating Only	,	·		en Paired Wi			
100 – 400%		Reco	mmended: 1	100 – 140%	Allowed: 1	00 – 400%	
Equipment Controls							
4.343 All equipment controls below have been included who	ere applicable	in the HV	AC Design.				
4.354 All heating and cooling systems serving a dwelling un				ithin the dwe	lling unit -whi	ch are not loc	cated on
4.354.1 Prescriptive Path: Dwelling unit thermostats are p							
4.365 Stair and elevator shaft vents shall be equipped with a building operation and are interlocked to open as required by	motorized da	mpers that			omatically clo	osed during r	ormal
4.376 Freeze protection systems, such as heat tracing of pil heaters shall include automatic controls capable of shutting Where heat tracing is specified for freeze protection, control required.	ping and hear off the syster	t exchange ns when pi	rs, including pe wall or ga	self-regulati arage / plenu	ım temperatu	res are abov	e 40°F.
4.387 Snow- and ice-melting systems shall include automat above 50°F and no precipitation is falling, and an automatic so that the potential for snow or ice accumulation is negligib	or manual co						
Hydronic Distribution Requirements – Applies to heat unit	ing or coolin	g systems	serving mo	re than one	dwelling □ N/	A	
4.398 All hydronic distribution requirements below have bee	n included w	here applic	able in the H	IVAC Design	١.		
4.4039 All terminal heating and cooling distribution equipmed distribution pump, so that heated or cooled fluid is not delive thermostat.							
4.410 Terminal units must be equipped with pressure independent	endent balan	cing valves	or pressure	independer	t control valv	es.	
4.424 Piping of a heating or cooling system (e.g., steam, ho ASHRAE 90.1-2007, Table 6.8.3. Construction documents r	nust account	for piping t	otal thicknes	ss including	required insu	lation when p	nce with passing
through planks or any other penetrations and shall specify the Heating System: Pipe size: inches Insulation thic			ispected bet Pipe size:	ore access i	s covered up Insulation t		inches
4.432 For circulating pumps serving hydronic heating or coceexceed efficiency standards for NEMA Premium™ motors. I	ling systems	with three-	phase moto	rs, 1 horse-p	ower or large	er, motors sh	all meet or
4.443 If a variable speed pumping system is installed, syste such as a minimum flow bypass valve or 3-way valves on speed pumping system is installed.	m designed t	o prevent "					
4.454 For shared boilers, chillers, and cooling towers, tempor clearly shown on the drawings. A complete sequence of opercondensing boilers, design return temperature is indicated as	erature and perations for al	ressure ga I systems i	ndicating red	commendation	ons for all set	points is prov	/ided. <u>For</u>



5. Dwelling Unit Duct Design (Comp	lete if heating or cooling	equipi	ment will be installed with	ducts; otherwis	se check "N/A".)	Designer Verified
						□ N/A
5.1 Duct system designed for the equipm Townhouses only: Duct system must			□ ACCA Manual D □ C D.	Other:		
5.2 Room-by-room design airflows docur	mented below (which shou	ıld mu	st sum to the mode with the	higher Design I	HVAC fan airflow).	<u>109, 4339, 44</u> 40
Name of the unit plan:	·		e of the unit plan:		·	
Design HVAC fan airflow: 4541		Desig	ın HVAC fan airflow: 4541			
	ng mode CFM		ng mode CFM	Heating mode		
9	ng mode		gn HVAC fan speed setting on mode	e.g., low, mediu Heating mode		
Design total external static pressure (corwith the higher airflow above): 4743	responding to the mode IWC	Desig	gn total external static press he higher airflow above): ⁴⁷	ure (correspond	ling to the mode	
Room Name	Design Airflow (CFM)		Room Name		Design Airflo	w (CFM)
1		1				
2		2				
3		3				
5		4 5				
6		6				
7		7				
8		8				
9		9				
10		10				
Total for all rooms			To	tal for all rooms		
6. Duct Quality Installation - Applies	s to Heating, Cooling, Ve	entilati	on, Exhaust, & Pressure E	Balancing Ducts	s, Unless Noted ir	Footnote
6.1 All-Applicable duct quality installation	requirements in 6.2 – 6.8	below	have been included where	applicable in th	e HVAC Design.	
6.2 Ductwork specified without kinks, sha	arp bends, compressions,	or exc	essive coiled flexible ductw	ork. <u>48</u> 44		
6.3 All supply and return ducts not in con	ditioned space, including	conne	ctions to trunk ducts, are ins	sulated to ≥ R-6	<u>49</u> 45	
6.3.1 Prescriptive Path: Dwelling unit Design.	ductwork meets the locati	ion and	d insulation requirements sp	ecified in the El	NERGY STAR MF	Reference
Dwelling Unit						
6.4 MERV 6+ filter(s) specified for each of supplied outdoor air passes through filter owner, or building maintenance staff. Filt supplied outdoor air designed to pass the control of the building specified such that the control of the supplied outdoor air designed to pass the control of the building specified such that the control of the supplied outdoor air designed to pass the control of the supplied outdoor air designed to pass the control of the supplied outdoor air passes through filter own the supplied outdoor air designed to pass the supplied outdoor air designed to be supplied outdoor air designed to be supplied to be supplied outdoor air designed to be supplied outdoor air designed to be supplied to be supplied outdoor air designed to be supplied to be supplied to be supplited outdoor air designed to be supplied to be supplied to be supp	r(s) prior to conditioning, a er access panel specified rough filter prior to condition that Rater-measured total	ind loc with a oning. duct le	ated to facilitate access & re gasket or comparable seali eakage is ≤ 4 CFM25 per 10	egular service by ing mechanism. 0 ft ² of CFA at r	y the occupant <u>. or</u> All return air and i ough-in or ≤ 8 CFN	building mechanically M25 per 100
ft² at final, or if there are no ducted return Townhouses only, Rater-measured duct	leakage to the outside is	≤4 CF	M25 per 100 ft² of CFA or ≤	40 CFM25. 5147		
6.6 <u>All bedrooms provided with transfer of</u> 150 CFM (as reported in Item 5.2) are specified to achieve a Rater-meas air handlers are operating. <u>Townhouses measured pressure differential ≥ - 3 Pa a</u>	pecified with any combinate ured pressure differential only: In addition, bedroom	ion of ≥ - 5 P	transfer grilles, jump ducts, Pa and ≤ 5 Pa with respect to	dedicated return the main body	n ducts, and/or und of the dwelling un	lercut doors it when all
Common Space and Central Exhaust						
6.7 Duct design specifies that all supply, joints, longitudinal seams, and duct wall		vork ar	nd all plenums serving comr	mon spaces sha	ıll be sealed at all t	ransverse
6.8 Central exhaust systems (that serve exceed 25% of exhaust fan flow at rough ductwork between the fan and the grilles	i-in (e.g., including trunks,					



ENERGY STAR Multifamily New Construction, Version 1 / 1.1 / 1.2 (Rev. 032)

Footnotes:

- 1. This report shall represent system design for all unique unit plans, common spaces, and where applicable, parking garages. The term 'common space' refers to any spaces in the building being certified that serve a function in support of the residential part of the building that is not part of a dwelling or sleeping unit. This includes spaces used by residents, such as corridors, stairs, lobbies, laundry rooms, exercise rooms, residential recreation rooms, and dining halls, as well as offices and other spaces used by building management, administration or maintenance in support of the residents. As an alternative, for dwelling units, project teamsdesigners may instead choose to complete a Single-Family New Homes National HVAC Design Report for each unique unit plan, if room-by-room loads are calculated using Unabridged ACCA Manual J v8. For those unit plans, Items 3.7 and 3.13 of this Report would still need to be completed. Sections 4 and 5 must be completed in either Design Report unless exempted by this Report. All other systems, including all systems serving common spaces, must be documented in this Design Report. This report is designed to meet ASHRAE 62.1-2010 or later / 2013, ASHRAE 62.2-2010 or later / 2013, and ANSI / ACCA's 5 QI-2015 protocol, thereby improving the performance of HVAC equipment in new multifamily buildings when compared to multifamily buildings built to minimum code. However, these features alone cannot prevent all ventilation, indoor air quality, and HVAC problems (e.g., those caused by a lack of maintenance or occupant behavior). Therefore, system designs documented through the use of this report are not a guarantee of proper ventilation, indoor air quality, or HVAC performance.
- 2. The term 'Rater' refers to the person(s) completing the third-party verification required for certification. The person(s) shall: a) be a Certified Rater, Approved Inspector, as defined by ANSI / RESNET / IECC Standard—301, or an equivalent designation as determined by a Home Certification Organization (HCO) or Multifamily Review Organization (MRO); and, b) have attended and successfully completed an EPA-recognized training class. See www.energystar.gov/mftraining.
- 3. As defined by ANSI / RESNET / ICC Std. 301-2019, a Dwelling Unit Mechanical Ventilation System is a ventilation system consisting of powered ventilation equipment such as motor-driven fans and blowers and related mechanical components such as ducts, inlets, dampers, filters and associated control devices that provides dwelling-unit ventilation at a known or measured airflow rate.
- 4. The dwelling-unit mechanical ventilation system shall have at least one supply or exhaust fan with associated ducts and controls. Local exhaust fans are allowed to be part of a dwelling-unit mechanical ventilation system. Designers may provide supplemental documentation as needed to document the system design.
- 5. In "Warm-Humid" climates as defined by 2009 IECC Figure 301.1 (i.e., CZ 1 and portions of CZ 2 and 3A below the white line), it is recommended, but not required, that equipment be specified with sufficient latent capacity to maintain indoor relative humidity at ≤ 60%.
- 6. Item 2.187 applies to any outdoor air inlet connected to a ducted return of the dwelling unit HVAC system, regardless of its intended purpose (e.g., for ventilation air, make-up air, combustion air). This Item does not apply to HVAC systems without a ducted return. For example, if an outdoor air inlet connected to a ducted return is used as a dedicated source of outdoor air for an exhaust ventilation system (e.g., bath fan), the outdoor airflow must be automatically restricted when the exhaust fan is not running and in the event of an override of the exhaust ventilation system.
 - In dwelling / sleeping units in multifamily buildings, but not townhouses, automatic restriction of airflow is exempted if a manual shutoff damper is used with a continuous exhaust ventilation system and is readily-accessible, labeled as the override, and not used as a balancing damper.
 - Note that a Rater will generally measure the ventilation rate at the highest HVAC fan speed applicable to ventilation mode (e.g., if the inlet only opens when the HVAC is in 'fan-only' mode, then test in this mode) to verify that it is ≤ 15 CFM or 15% above design value. If the inlet has a motorized damper that only opens when the local mechanical kitchen exhaust is turned on, then testing is not required. As an alternative, measurement of the outdoor airflow can be waived if a Constant Airflow Regulating (CAR) damper with a manufacturer-specified maximum flow rate no higher than 15 CFM or 15% above the ventilation design value is installed on the inlet.
- Airflow design rates and run-times shall be determined using ASHRAE 62.2-2010 or later. Designers are permitted, but not required, to use
 published addenda and/or the 2013more recent versions editions of the standard to assess compliance. The year of the standard that is used
 shall be listed in the space provided.
- 8. Airflow design rates shall be determined using ASHRAE 62.1-2010 or later. Designers are permitted, but not required, to use published addenda and/or more recent the 2013 versions editions of the standard to assess compliance. The year of the standard that is used shall be listed in the space provided.
- 8-9. For permits on or before 01/01/2024, where outdoor air is supplied to a common space via a PTAC or PTHP, in lieu of measurement, the design CFM shall meet or exceed the ventilation rates required by ASHRAE 62.1-2010 and the space served by the PTAC or PTHP shall have at least one operable window. For permits after 01/01/2024, both the runtime and measurement of outdoor air through these systems will be required to demonstrate compliance with ASHRAE 62.1-2010 or alternative ventilation system specified (e.g., ducted supply).
- 9-10. If the tables provided cannot accommodate all the unit plans, spaces, or systems in the project building, use the tables in Appendix A to supplement the Design Report.
- 11. List each individual common space separate from other spaces, such that when reporting airflow for Items 2.8 and 2.9, compliance for each space can be demonstrated. For example, list an office space separate from a community room, even if these spaces are served by the same system and even if the outdoor air rates required are the same. Similarly, where a space is repeated in the building, such as a corridor, report each space by floor (e.g., FL1 Corridor, FL2 Corridor). Rather than list these values in this report, as an alternative, the HVAC Designer is permitted to submit the values in a separate document or file.
- 10.12. In addition, consult manufacturer requirements to ensure return air temperature requirements are met.
- 41.13. Dwelling-unit mechanical ventilation fans shall be rated for sound at no less than the airflow rate in Item 2.7. Fans exempted from this requirement include HVAC air handler fans, remote-mounted fans, and intermittent fans rated ≥ 400 CFM. To be considered for this exemption, a remote-mounted fan must be mounted outside the habitable spaces, bathrooms, toilets, and hallways and there shall be ≥ 4 ft. ductwork between the fan and intake grill. Per ASHRAE 62.2-2010, habitable spaces are intended for continual human occupancy; such space generally includes areas used for living, sleeping, dining, and cooking but does not generally include bathrooms, toilets, hallways, storage areas, closets, or utility rooms.
- 12-14. Note that the 'fan-on' setting of a thermostat would not be an acceptable controller because it would continuously operate the HVAC fan.

Revised 0911/212/20220 Page 6 of 13



National HVAC Design Report 1

ENERGY STAR Multifamily New Construction, Version 1 / 1.1 / 1.2 (Rev. 032)

- 13.15. Bathroom fans with a rated flow rate ≥ 500 CFM and heat/energy recovery ventilation fans are exempted from the requirement to be ENERGY STAR certified.
- 44.16. Without proper maintenance, ventilation air inlet screens often become filled with debris. Therefore, EPA recommends, but does not require, that these ventilation air inlets be located so as to facilitate access and regular service by the building maintenance staff.
- 45.17. Continuous bathroom local mechanical exhaust fans shall be rated for sound at no less than the design airflow rate. Intermittent bathroom and both intermittent and continuous kitchen local mechanical exhaust fans are recommended, but not required, to be rated for sound at no less than the design airflow rate. Per ASHRAE 62.2-2010, an exhaust system is one or more fans that remove air from the building, causing outdoor air to enter by ventilation inlets or normal leakage paths through the building envelope (e.g., bath exhaust fans, range hoods, clothes dryers). Per ASHRAE 62.2-2010, a bathroom is any room containing a bathtub, shower, spa, or similar source of moisture.
- 46.18. An intermittent mechanical exhaust system, where provided, shall be designed to operate as needed by the occupant. Control devices shall not impede occupant control in intermittent systems.
- 47.19. Kitchen volume shall be determined by drawing the smallest possible rectangle on the floor plan that encompasses all cabinets, pantries, islands, peninsulas, ranges / ovens, and the kitchen exhaust fan, and multiplying by the average ceiling height for this area. In addition, the continuous kitchen exhaust rate shall be ≥ 25 CFM, per 2009 IRC Table M1507.3, regardless of the rate calculated using the kitchen volume. Cabinet volume shall be included in the kitchen volume.
- 48.20. While not required, the prescriptive duct sizing requirements in Table 5.3 of ASHRAE 62.2-2010 are recommended to be used for kitchen exhaust fans based upon the rated airflow of the fan at 0.25 IWC.
- 49.21. As an alternative, dwelling units are permitted to use a continuous kitchen exhaust rate of 25 CFM per 2009 IRC Table M1507.3, if they are either a) PHIUS+ or PHI certified, or b) provide both dwelling unit ventilation and local mechanical kitchen exhaust using a balanced system, and have a Rater-verified whole-building infiltration rate ≤ 1.0 ACH50 or ≤ 0.05 CFM50 per ft² of Enclosure Area. 'Enclosure Area' is defined as the area of the surfaces that bound the volume being pressurized / depressurized during the test.
- 20.22. All intermittent kitchen exhaust fans must be capable of exhausting at least 100 CFM. In addition, if the fan is not part of a vented range hood or appliance-range hood combination (i.e., if the fan is not integrated with the range), then it must also be capable of exhausting ≥ 5 ACH, based on the kitchen volume.
- 21.23. For continuous system operation, the lower rate may be used. Otherwise, use the higher rate. Commercial kitchens shall be designed to provide a minimum continuous rate of 0.70 cfm/ft².
- 22.24. As an alternative, for a toilet room intended to be occupied by one person at a time, a minimum continuous rate of 25 cfm is permitted.
- 23._____This section / item applies to split air conditioners, unitary air conditioners, air-source heat pumps, and water-source (i.e., geothermal) heat pumps up to 65 kBtuh with forced-air distribution systems and to furnaces up to 225 kBtuh with forced-air distribution system serving individual dwelling units. Forced-air distribution systems are those that supply air through ductwork exceeding 0 ft. in length. This section / item is recommended, but not required for non-ducted systems, such as non-ducted mini-splits, multi-splits, PTHP's, or PTAC's.
- 24.26. Select "2013 / 2017 ASHRAE Fundamentals" if using Chapter 17 of the 2013 or 2017 ASHRAE Handbook of Fundamentals. Select "Other per AHJ" if the Authority Having Jurisdiction where the unit will be certified mandates the use of a load calculation methodology other than Unabridged ACCA Manual J v8 or 2013 or 2017 ASHRAE Handbook of Fundamentals.
- 25.27. Check the box for "unit-specific design" if the design was created for the specific plan configuration (i.e., elevation, option, orientation, and county) of the unit to be certified. Check the box for "group design" if designs were created for unit plans that are repeated throughout the project +building with potentially different configurations (i.e., different elevations and/or orientations). Check the box for "worst-case design" if loads for the unit with the largest heat gain in the project +building are less than 18 kBtuh and are being used to represent all other units. Only one box may be checked. Regardless of the box checked, the system design as documented on this HVAC Design Report must fall within the following tolerances for the unit to be certified:
 - Item 3.4: The outdoor design temperature used in loads are within the limits defined at www.energystar.gov/hvacdesigntemps.
 - Item 3.6: The number of occupants used in loads is within ± 2 of the dwelling unit to be certified.
 - Item 3.7: Total occupant gains used in loads shall not exceed 645 Btuh per occupant.
 - Item 3.8: The conditioned floor area used in loads is between 100 ft² smaller and 300 ft² larger than the dwelling unit to be certified.
 - Item 3.9: The window area used in loads is between 15 ft² smaller and 60 ft² larger than the dwelling unit to be certified, or for dwelling units with > 500 ft² of window area, between 3% smaller and 12% larger.
 - Item 3.10: The predominant window SHGC is within 0.1 of the predominant value in the dwelling unit to be certified.
 - Item 3.12: The mechanical ventilation rate used in loads is the same as the value in Section 2a for the given unit plan.
 - Item 3.13: The sum of the internal gains associated with lighting and appliances used in loads shall not exceed 3,600 Btuh.
 - Items 3.15 & 3.17: The sensible & total heat gain are documented for the orientation configuration of the dwelling unit to be certified.
 - Item 4.189: The cooling sizing % is within the cooling sizing limit selected.

Provide the National HVAC Design Report to the party you are providing these design services to (i.e., a builder / developer, Functional Testing Agent (FT Agent), and/or MEP / credentialed HVAC contractor) and to the Rater. The report is only required to be provided once per project / building. As long as a report has been provided that falls within these tolerances for the units to be certified, no additional work is required. However, if no report falls within these tolerances or if any aspect of the system design changes, then an additional report will need to be generated prior to certification.

Visit www.energystar.gov/hvacdesigntools for a tool to assist with group designs and for more information.

26._28. For each unique unit floorplan, document the loads for the configuration (e.g., level, orientation) that the dwelling unit might be built in. For example, if a unit plan will only be built in a specific level and orientation (e.g., top-floor, facing South), then the designer only needs to document the loads for this one configuration. To determine whether a unit floorplan is "unique", the guidance in ANSI 301-2019, Section 5.1.4.4.1 may be

Revised 0911/212/20220



ENERGY STAR Multifamily New Construction, Version 1 / 1.1 / 1.2 (Rev. 032)

followed. Orientation represents the direction that the front door of the dwelling unit is facing. In Section 4, to calculate Cooling sizing % for each configuration of each unique floorplan, the same system may need to be duplicated in multiple columns.

- 27.29. Visit www.energystar.gov/hvacdesigntemps for the maximum cooling season design temperature and minimum heating season design temperature permitted for ENERGY STAR. For "County & State, or US Territory, selected", select the County and State or US Territory (i.e., Guam, Northern Mariana Islands, Puerto Rico, or US Virgin Islands), where the unit is to be certified. The same design report is permitted to be used in other counties, as long as the design temperature limits in those other counties meet or exceed the cooling and heating season temperature limits for the county selected. For example, if Fauquier County, VA, is used for the load calculations, with a 1% cooling temperature limit of 93°F, then the same report could be used in Fairfax County (which has a higher limit of 94°F) but not in Arlington County (which has a lower limit of 92°F). If a jurisdiction-specified design temperature is used that exceeds the limit in the ENERGY STAR Single-Family New Homes Design Temperature Limit Reference Guide, designers must submit a Design Temperature Exception Request. Visit www.energystar.gov/hvacdesigntemps for a copy of this form.
- 28.30. To determine the number of occupants among all HVAC systems in the dwelling unit, calculate the number of bedrooms, as defined below, and add one. This number of occupants must be within ± 2 of the dwelling unit to be certified.

A bedroom is defined by ANSI / RESNET / ICC Standard 301-2014 as a room or space 70 ft² or greater size, with egress window and closet, used or intended to be used for sleeping. A "den", "library", or "home office" with a closet, egress window, and 70 ft² or greater size or other similar rooms shall count as a bedroom, but living rooms and foyers shall not.

An egress window, as defined in 2009 IRC section R310, shall refer to any operable window that provides for a means of escape and access for rescue in the event of an emergency. The egress window definition has been summarized for convenience. The egress window shall:

- have a sill height of not more than 44 inches above the floor; AND
- have a minimum net clear opening of 5.7 ft²; AND
- have a minimum net clear opening height of 24 in.; AND
- have a minimum net clear opening width of 20 in.; AND
- be operational from the inside of the room without the use of keys, tools or special knowledge.
- 29.31. The difference between the Conditioned Floor Area (CFA) used in the design and the actual dwelling unit to be certified must fall within the tolerance specified in Footnote 22, as verified by a Rater. Be advised, the Rater will calculate CFA using the definition in ANSI / RESNET / ICC Standard-301-2019, which defines this value, in part, as the floor area of the Conditioned Space Volume within a building or Dwelling Unit, not including the floor area of attics, crawlspaces, and basements below air sealed and insulated floors. See www.codes.iccsafe.org/content/chapter/16185/ for the complete definition.
- 30.32. The difference between the window area used in the design and the actual dwelling unit to be certified must fall within the tolerance specified in Footnote 22, as verified by a Rater. Be advised, the Rater will calculate window area using the on-site inspection protocol provided in Normative Appendix B of ANSI / RESNET / ICC Standard 301-2019, which instructs the Rater to measure the width and height of the rough opening for the window and round to the nearest inch, and then to use these measurements to calculate window area, rounding to the nearest tenth of a square foot. See www.codes.iccsafe.org/content/chapter/16191/ for the complete protocol.
- 31.33. "Predominant" is defined as the SHGC value used in the greatest amount of window area in the dwelling unit.
- 32.34. Infiltration rate shall use "Tight" values for the cooling season infiltration rate and "Tight" values for the heating season infiltration rate, as defined by Table 5A or 5B of ACCA Manual J, Eighth Edition, Version Two. Alternatively, infiltration rate shall not exceed 0.24 air changes per hour.
- 35. Equipment shall be selected using the maximum total heat gain and the total heat loss in Section 3 per ACCA Manual S, Second Edition, except that cooling ranges above ACCA Manual S limits are temporarily allowed, per Item 4.19, and heating ranges above ACCA Manual S limits are allowed where heating and hot water are provided by the same equipment or where standby equipment is needed for redundancy, but only operate when the primary equipment is not operating. For equipment outside the scope of ACCA Manual S, "Other" may be indicated and the equipment sizing approach listed in the space provided.
- 33.36. These requirements apply to systems that provide primary space heating and cooling. Electric resistance limitations do not apply to systems dedicated to heating outdoor air supplied by a mechanical ventilation system, as long as the space served is primarily heated by a non-electric-resistance system that meets the efficiency requirements noted in Exhibit X. Electric resistance limitations apply to garages, but do not apply to heated plenums meeting Item 4.37, or stairwells where automatic thermostatic controls prevent operation above 50°F.
- 34.37. If an AHRI Reference # is not available, OEM-provided documentation shall be attached with the rated efficiency. For residential split air conditioners and heat pumps, the rated efficiency shall be for of the specific combination of indoor and outdoor components of the air conditioner or heat pump, along with confirmation that the two components are designed to be used together. If the AHRI Reference # is reported in Item 4.10 (e.g., heat pumps), the AHRI Reference # does not need to be listed again in Item 4.27.
- 35.38. The full system Capacity will be listed as the capacity at design conditions, from OEM expanded performance data, shall be listed and shall include the capacity of all systems providing space cooling to the dwelling unit. For two-speed or variable-speed equipment, the full system capacity shall reflect the capacity at the maximum available compressor speed or when the compressor operates at the AHRI rating test speed, respectively.
- 36.39. Per ACCA Manual S, Second Edition, if the load sensible heat ratio is ≥ 95% and the HDD / CDD ratio is ≥ 2.0, then the Climate is Condition B, otherwise it is Condition A.
- 40. As an alternative for low-load dwelling units, a system match-up including a single-speed compressor with a total capacity ≤ 20 kBtuh is permitted to be used in spaces with a total cooling load ≤ 15 kBtuh. A system match-up including a two-speed or variable-speed compressor with a total capacity ≤ 25 kBtuh is permitted to be used in spaces with a total cooling load ≤ 18 kBtuh.
- 37.41. The full system capacity shall be listed for the heating system. For two-stage and modulating furnaces, the full system capacity shall reflect the maximum output available. For shared boilers, the full system capacity may exclude standby equipment needed for redundancy.
- 38.42. Per the 2009 International Mechanical Code, a direct-vent furnace or boiler is one that is constructed and installed so that all air for combustion is derived from the outdoor atmosphere and all flue gases are discharged to the outside atmosphere; a mechanical draft system is

Revised 0911/212/20220 Page 8 of 13



National HVAC Design Report 1

ENERGY STAR Multifamily New Construction, Version 1 / 1.1 / 1.2 (Rev. 032)

a venting system designed to remove flue or vent gases by mechanical means consisting of an induced draft portion under non-positive static pressure or a forced draft portion under positive static pressure; and a natural draft system is a venting system designed to remove flue or vent gases under non-positive static vent pressure entirely by natural draft. Naturally drafted equipment is only allowed if located in a space outside the pressure boundary, where the envelope assemblies separating it from conditioned space are insulated and air-sealed. For mechanically drafted boilers, make-up air sources must be mechanically closed when the boiler is not in operation.

- 39.43. Designers may provide supplemental documentation with room-by-room and total design airflows in lieu of completing Item 5.52. Sample supplemental documentation can be found at www.energystar.gov/hvacdesigntools.
- 40.44. Orientation-specific room-by-room design airflows are recommended, but not required, to distribute airflow proportional to load, thereby improving comfort and efficiency. While air-balancing of supply registers and return grilles is not required to be completed as part of HVAC Functional Testing, it is recommended that ducted HVAC systems be designed such that they can be balanced in the field (i.e. provide proper access to any and all balancing dampers, provide ducting and grille layouts such that accurate air measurements can be taken).
- 41.45. Design HVAC fan airflow is the design airflow for the blower in CFM, as determined using the manufacturer's expanded performance data. The Functional Testing Agent is required to measure the HVAC fan airflow using the mode with the higher airflow, within ± 15% of design.
- 42.46. Design HVAC fan speed setting is the fan speed setting on the control board (e.g., low, medium, high) that corresponds with the Design HVAC fan airflow.
- 43.47. Design total external static pressure is the pressure corresponding to the Design HVAC fan airflow, inclusive of external components (e.g., evaporator coil, whole-house humidifier, or ≥ MERV 6 filter).
- 44.48. Kinks are to be avoided and are caused when ducts are bent across sharp corners such as framing members. Sharp bends are to be avoided and occur when the radius of the turn in the duct is less than one duct diameter. Compression is to be avoided and occurs when flexible ducts in unconditioned space are installed in cavities smaller than the outer duct diameter and ducts in conditioned space are installed in cavities smaller than inner duct diameter. Ducts shall not include coils or loops except to the extent needed for acoustical control.
- 45.49. Item 6.3 does not apply to ducts that are a part of local mechanical exhaust or exhaust-only dwelling-unit ventilation systems. EPA recommends, but does not require, that all metal ductwork not encompassed by Section 6 (e.g., exhaust ducts, duct boots, ducts in conditioned space) also be insulated and that insulation be sealed to duct boots to prevent condensation.
- 46.50. Item 6.5 generally applies to the ducts of space heating, space cooling, and Dewelling Uunit Mmechanical Vventilation Seystems. However, visual inspection is permitted in lieu of testing for the following system types: 1) a Dewelling Uunit mMechanical vventilation Seystem not connected to the space heating or space cooling system, regardless of the number of dwelling units it serves; 2) a space heating or space cooling system for which the ducts and air handler are in conditioned space and the total supply duct length of the system, including all supply trunks and branches, is ≤ 10 ft; and 3) a space heating or space cooling system that serves more than one dwelling unit. In such cases, a Rater shall visually verify that all seams and connections are sealed with mastic or metal tape and all duct boots are sealed to floor, wall, or ceiling using caulk, foam, or mastic tape.
- 47.51. Duct leakage shall be determined and documented by a Rater in accordance with ANSI / RESNET / ICC Std. 380. Leakage limits shall be assessed on a per-system, rather than per-dwelling unit, basis. For a duct system with one or two returns, the total Rater-measured duct leakage is permitted to be the greater of ≤ 4 CFM25 per 100 ft² of CFA or ≤ 40 CFM25 at 'rough-in' or the greater of ≤ 8 CFM25 per 100 ft² of CFA or ≤ 8 CFM25 at 'final'. For a duct system with three or more returns, the total Rater-measured duct leakage is permitted to be the greater of ≤ 6 CFM25 per 100 ft² of CFA or ≤ 60 CFM25 at 'rough-in' or the greater of ≤ 12 CFM25 per 100 ft² of CFA or ≤ 120 CFM25 at 'final'. For a duct system without any ducted returns, the total Rater-measured duct leakage is permitted to be the greater of ≤ 3 CFM25 per 100 ft² of CFA or ≤ 30 CFM25 at 'rough-in' or the greater of ≤ 6 CFM25 per 100 ft² of CFA or ≤ 60 CFM25 at 'final' and, the Rater-measured pressure difference between the space containing the air handler and the conditioned space, with the air handler running at high speed, is ≤ 5 Pa. For systems > 1 ton, increase by 1 Pa per half ton.
- 48-52. For the purpose of computing leakage allowance, exhaust fan flow shall be the lesser of the rated fan flow and at rough-in, 133% of the sum of the design exhaust airflow of the dwelling units that are exhausted by that central fan or at final, 143% of the sum of the design exhaust airflow of the dwelling units that are exhausted by that central fan. Measured fan flow (either at the fan itself or the total airflow measured from all exhaust grilles served by the fan) may be used in lieu of the rated fan flow to determine the leakage allowance. This test is not required of central exhaust systems serving clothes dryers but is required for the central exhaust portion of balanced systems such as HRVs and ERVs.



ENERGY STAR Multifamily New Construction, Version 1 / 1.1 / 1.2 (Rev. 032)

Appendix A – Supplementary tables for Section 2 and 3

2a. Dwelling Unit & Common Space Mechanical Ven	tilation Des	ign ^{4, 5}						
List unique unit plan for which 62.2 ventilation rates were calculated in the spaces to the right:								
2.4 # of bedrooms:								
2.5 Square footage:								
2.6 Ventilation airflow rate required by ASHRAE 62.2:								
2.7 Ventilation airflow rate designed:								
2.7.1 If applicable, run-time per cycle (minutes):								
2.7.2 If applicable, cycle time (minutes):								
List common space for which 62.1 ventilation rates were calculated in the spaces to the right:								
2.8 Ventilation airflow rate required by ASHRAE 62.1:								
2.9 Ventilation airflow rate designed:								
-		•	•				,	
System Type & Controls:								
List Ventilation System ID in the spaces to the right:								
2.10 Specified system type: (e.g., supply, exhaust, balanced, ERV, HRV)								
2.11 Specified system type: (e.g., in-unit, central)								
2.112 Manufacturer:								
2.123 Model Number:								
2.13 # installed in the building:								
2.14 Spaces each fan serves (i.e., single, multiple)								
2.145 Area / space(s) that system serves: (e.g., Unit A kitchens, corridor, community room)								
2.165 Specified control location: (e.g., Master bath, utility):								
		•	•			•	•	
3. Heating & Cooling Loads								
Dwelling Unit Heating & Cooling Loads (only required	for ducted	split AC, ur	itary AC	C, ASHP, V	VSHP, GSH	IP, and fu	rnaces) 24	²² □ N/A
List the unit plan for which Loads were calculated:								
3.5 Location of Unit: top, mid, bottom, corner, interior								
3.6 Number of occupants used in loads: 2725, 3028								
3.7 Total occupant gains (Btuh): 2725								
3.8 Conditioned floor area used in loads: 2725, 3129								
3.9 Window area used in loads: ^{25, 30}								
3.10 Predominant window SHGC used in loads: ^{25, 31}								
3.11 Infiltration (ACH / ACH50) used in loads: ³²								
3.12 Mechanical ventilation (CFM) used in loads:								
3.13 Non-occupant Internal gains (appliance, equipment and lighting) used in loads (Btuh): ²⁵								
3.14 <u>Door Oo</u> rientation (N, NE, E, SE, S, SW, W, NW): ²⁶								1
3.15 Sensible Heat Gain At Design Conditions (kBtuh): ²⁵	<u> </u>							1
3.16 Latent Heat Gain At Design Conditions (kBtuh):								1
3.17 Total Heat Gain at Design Conditions (kBtuh): ²⁵	<u> </u>						1	1
3.18 Total Heat Loss at Design Conditions (kBtuh):	<u> </u>						j	



ENERGY STAR Multifamily New Construction, Version 1 / 1.1 / 1.2 (Rev. 032)

Appendix A – Supplementary tables for Section 3

3.19 Common Space Heating &	Cooling Loads			
Common Space Name:	Design Conditions: Total Heat Gain:	(kBtuh)	Total Heat Loss:	(kBtuh)
Common Space Name:	Design Conditions: Total Heat Gain:	(kBtuh)	Total Heat Loss:	(kBtuh)
Common Space Name:	Design Conditions: Total Heat Gain:	(kBtuh)	Total Heat Loss:	(kBtuh)
Common Space Name:	Design Conditions: Total Heat Gain:	(kBtuh)	Total Heat Loss:	(kBtuh)
Common Space Name:	Design Conditions: Total Heat Gain:	(kBtuh)	Total Heat Loss:	(kBtuh)
Common Space Name:	Design Conditions: Total Heat Gain:	(kBtuh)	Total Heat Loss:	(kBtuh)
Common Space Name:	Design Conditions: Total Heat Gain:	(kBtuh)	Total Heat Loss:	(kBtuh)
Common Space Name:	Design Conditions: Total Heat Gain:	(kBtuh)	Total Heat Loss:	(kBtuh)
Common Space Name:	Design Conditions: Total Heat Gain:	(kBtuh)	Total Heat Loss:	(kBtuh)
Common Space Name:	Design Conditions: Total Heat Gain:	(kBtuh)	Total Heat Loss:	(kBtuh)
Common Space Name:	Design Conditions: Total Heat Gain:	(kBtuh)	Total Heat Loss:	(kBtuh)
Common Space Name:	Design Conditions: Total Heat Gain:	(kBtuh)	Total Heat Loss:	(kBtuh)
Common Space Name:	Design Conditions: Total Heat Gain:	(kBtuh)	Total Heat Loss:	(kBtuh)
Common Space Name:	Design Conditions: Total Heat Gain:	(kBtuh)	Total Heat Loss:	(kBtuh)
Common Space Name:	Design Conditions: Total Heat Gain:	(kBtuh)	Total Heat Loss:	(kBtuh)
Common Space Name:	Design Conditions: Total Heat Gain:	(kBtuh)	Total Heat Loss:	(kBtuh)
Common Space Name:	Design Conditions: Total Heat Gain:	(kBtuh)	Total Heat Loss:	(kBtuh)
Common Space Name:	Design Conditions: Total Heat Gain:	(kBtuh)	Total Heat Loss:	(kBtuh)
Common Space Name:	Design Conditions: Total Heat Gain:	(kBtuh)	Total Heat Loss:	(kBtuh)
Common Space Name:	Design Conditions: Total Heat Gain:	(kBtuh)	Total Heat Loss:	(kBtuh)

3.20 Building Heating & Co	3.20 Building Heating & Cooling Loads (only required when shared systems such as central boilers or chillers are specified)									
System Name:	Design Conditions: Total Heat Gain:	_(kBtuh)	Total Heat Loss:(kBtuh)							
System Name:	Design Conditions: Total Heat Gain:	_(kBtuh)	Total Heat Loss:(kBtuh)							
System Name:	Design Conditions: Total Heat Gain:	_(kBtuh)	Total Heat Loss:(kBtuh)							
System Name:	Design Conditions: Total Heat Gain:	_(kBtuh)	Total Heat Loss:(kBtuh)							

Revised <u>09</u>41/<u>2</u>42/202<u>2</u>0 Page 11 of 13



ENERGY STAR Multifamily New Construction, Version 1 / 1.1 / 1.2 (Rev. 032)

Appendix A – Supplementary tables for Section 4

4. Heating & Cooling Equipment Se	election								
Cooling Equipment (Complete all a (columns), identical data is not require								e spaces	
List Cooling Equipment ID in the spaces duplicating as needed for each uni									
4.4 Equipment type: (PTAC / AC, Chille WLHP / GSHP / ASHP / VRF)	r / CT, PTHP /								
4.5 Area / Space(s) that system serves:									
4.6 Chiller / condenser / outdoor unit ma	anufacturer:								
4.7 Chiller / condenser / outdoor unit mo	odel #:								
4.8 Evaporator / indoor unit manufactur	er:								
4.9 Evaporator / indoor unit model #:									
4.10 AHRI reference #: ³⁴									
4.11 AHRI-Listed efficiency:									
4.12 Evaporator fan type: PSC, ECM / I	vaporator fan type: PSC, ECM / ICM Other:								
4.13 Compressor speed: Single, Two, \									
4.14 Turn down ratio (for variable speed									
4.15 Latent capacity at design condition	ns (kBtuh): 35								
4.16 Sensible capacity at design condit	ions (kBtuh): 35								
4.17 Total capacity at design conditions	(kBtuh): ³⁵								
4.18 Cooling sizing % = Total capacity (by Total Heat Gain (Item 3.17) of space									
4.19 Meets cooling sizing limit: (A, B, C	, D or N/A) ²³ . 25								
4.20 If "B", list Load sensible heat ratio heat gain (Item 3.15) / Max. total heat g									
4.21 If "B", calculate HDD / CDD ratio: 3									
			Compre	ssor Type	(Per Item 4	·.13)			
Equipment Type & Climate Condition	Single-Spe	eed		Two-Speed		V	ariable-Speed	I	
A: For Cooling-Only Equipment or For Cooling Mode of Heat Pump in Condition A Climate ³⁶	Recommended Allowed: 90		Recommended: 90 – 120% Allowed: 90 – 140%			Recommended: 90 – 130% Allowed: 90 – 160%			
B: For Cooling Mode of Heat Pump in Condition B Climate ³⁶	90% - 100%, pl	us 15 kBtuh	90% - 100%, plus 15 kBtuh		15 kBtuh	90% -	5 kBtuh		
C: For low-load spaces (≤15 kBtuh) ³⁷	≤ 20 kB	Btuh							
D: For low-load spaces (≤18 kBtuh) ³⁷				≤ 25 kBtu	h		≤ 25 kBtuh		
Heating Equipment (Complete all a multiple spaces (columns), identical c check "N/A".)	lata is not required							□ N/A	
List Heating Equipment ID in the space duplicating as needed for each unique s									
4.22 Electric equipment type: PTHP, W VRF, Boiler, Furnace, Electric Res									
4.23 Gas Equipment type: HW PTAC / PTAC, Boiler, Furnace	fan coil, Gas-Fired								
4.24 Area / Space(s) that system serve	s:								
4.25 Manufacturer:									
4.26 Model Number:									
24.27 AHRI reference #: 34									
4.2 <mark>78</mark> Listed efficiency:									
4.2 <u>9</u> 8 Equipment output capacity (kBtul	า):								
4.3029 Air-source heat pump output ca	pacity (kBtuh)								
4.3 <u>10</u> Type of Venting: Natural Draft, M Direct Vent ³⁸	echanically Drafted,								



ENERGY STAR Multifamily New Construction, Version 1 / 1.1 / 1.2 (Rev. 032)

	,	J	,	0.0 ,	,	(<u>-</u>
4.324 Furnace heating sizing % = Total capacity (Item 4.289) divided by Total Heat Loss of space(s) in Item 4.24:							
4.332 Meets furnace sizing limit: (A, B, C, or N/A) 23							
A: For low-load spaces	(≤ 10 kBtuh)	, furnace οι	utput capaci	ty is ≤ 40 kBt	uh		
B: When Used for Heating Only			C: Whe	en Paired Wi	th Cooling		
100 – 400%		Reco	mmended: 1	00 – 140%	Allowed: 1	00 – 400%	

Appendix A – Supplementary tables for Section 5

5. Dwelling-Unit Duct Design								
5.2 Room-by-room design airflows docur	mented below (which shou	ld <u>mus</u>	t sum to the mode with the higher Desi	gn HVAC fan airflow). ^{9, 39, 40}				
Name of the unit plan:		Name	of the unit plan:					
Design HVAC fan airflow: 41			n HVAC fan airflow: ⁴¹					
	ng mode CFM		ng mode CFM Heating m					
Design HVAC fan speed setting (e.g., lov	w, medium, high): 42		n HVAC fan speed setting (e.g., low, m					
	ng mode		ng mode Heating m					
Design total external static pressure (cor with the higher airflow above): 43	responding to the mode IWC		n total external static pressure (corresp ne higher airflow above): ⁴³	•				
Room Name	Design Airflow (CFM)	willi li	Room Name					
1	Design Airliow (Crivi)	1	Room name	Design Airflow (CFM)				
2		2						
3		3						
4		4						
5		5						
6		6						
7		7						
8		8						
9		9						
10		10						
Total for all rooms			Total for all roo	ms				
Name of the unit plan:		Name	of the unit plan:					
D : 10/40 f : 6 41			Design HVAC fan airflow: ⁴¹					
Design HVAC fan airflow: 41								
Cooling mode CFM Heating	ng mode CFM	Coolir	ng mode CFM Heating m					
Cooling mode CFM Heatin Design HVAC fan speed setting (e.g., lov	w, medium, high): ⁴²	Coolir Desig	ng mode CFM Heating m n HVAC fan speed setting (e.g., low, m	edium, high): ⁴²				
Cooling mode CFM Heatin Design HVAC fan speed setting (e.g., low Cooling mode Heatin	w, medium, high): ⁴² ng mode	Coolir Desig Coolir	ng mode CFM Heating m n HVAC fan speed setting (e.g., low, m ng mode Heating m	edium, high): ⁴² ode				
Cooling mode CFM Heatin Design HVAC fan speed setting (e.g., lov	w, medium, high): ⁴² ng mode	Coolir Desig Coolir Desig	ng mode CFM Heating m n HVAC fan speed setting (e.g., low, m	edium, high): ⁴² ode onding to the mode				
Cooling mode CFM Heatin Design HVAC fan speed setting (e.g., low Cooling mode Heatin Design total external static pressure (cor	w, medium, high): ⁴² ng mode responding to the mode	Coolir Desig Coolir Desig with th	ng mode CFM Heating m n HVAC fan speed setting (e.g., low, m ng mode Heating m n total external static pressure (corresp	edium, high): ⁴² ode onding to the mode				
Cooling mode CFM Heatin Design HVAC fan speed setting (e.g., low Cooling mode Heatin Design total external static pressure (corwith the higher airflow above): 43 Room Name 1	w, medium, high): ⁴² ng mode responding to the mode	Coolir Desig Coolir Desig with th	ng mode CFM Heating m n HVAC fan speed setting (e.g., low, m ng mode Heating m n total external static pressure (corresp ne higher airflow above): 43 IW	edium, high): ⁴² ode onding to the mode				
Cooling mode CFM Heating Design HVAC fan speed setting (e.g., low Cooling mode Heating Design total external static pressure (corwith the higher airflow above): 43 Room Name 1	w, medium, high): ⁴² ng mode responding to the mode	Coolir Desig Coolir Desig with th	ng mode CFM Heating m n HVAC fan speed setting (e.g., low, m ng mode Heating m n total external static pressure (corresp ne higher airflow above): 43 IW	edium, high): ⁴² ode onding to the mode				
Cooling mode CFM Heating Design HVAC fan speed setting (e.g., low Cooling mode Heating Design total external static pressure (corwith the higher airflow above): 43 Room Name 1	w, medium, high): ⁴² ng mode responding to the mode	Coolir Desig Coolir Desig with th	ng mode CFM Heating m n HVAC fan speed setting (e.g., low, m ng mode Heating m n total external static pressure (corresp ne higher airflow above): 43 IW	edium, high): ⁴² ode onding to the mode				
Cooling mode CFM Heating Design HVAC fan speed setting (e.g., low Cooling mode Heating Design total external static pressure (corwith the higher airflow above): 43 Room Name 1	w, medium, high): ⁴² ng mode responding to the mode	Coolir Desig Coolir Desig with th	ng mode CFM Heating m n HVAC fan speed setting (e.g., low, m ng mode Heating m n total external static pressure (corresp ne higher airflow above): 43 IW	edium, high): ⁴² ode onding to the mode				
Cooling mode CFM Heatin Design HVAC fan speed setting (e.g., low Cooling mode Heatin Design total external static pressure (corwith the higher airflow above): 43 Room Name 1	w, medium, high): ⁴² ng mode responding to the mode	Coolir Desig Coolir Desig with th	ng mode CFM Heating m n HVAC fan speed setting (e.g., low, m ng mode Heating m n total external static pressure (corresp ne higher airflow above): 43 IW	edium, high): ⁴² ode onding to the mode				
Cooling mode CFM Heatin Design HVAC fan speed setting (e.g., lov Cooling mode Heatin Design total external static pressure (cor with the higher airflow above): 43 Room Name 1 2 3 4 5 6	w, medium, high): ⁴² ng mode responding to the mode	Coolir Desig Coolir Desig with th 1 2 3 4 5	ng mode CFM Heating m n HVAC fan speed setting (e.g., low, m ng mode Heating m n total external static pressure (corresp ne higher airflow above): 43 IW	edium, high): ⁴² ode onding to the mode				
Cooling mode CFM Heating Design HVAC fan speed setting (e.g., low Cooling mode Heating Design total external static pressure (corwith the higher airflow above): 43 Room Name 1	w, medium, high): ⁴² ng mode responding to the mode	Coolir Desig Coolir Desig with the	ng mode CFM Heating m n HVAC fan speed setting (e.g., low, m ng mode Heating m n total external static pressure (corresp ne higher airflow above): 43 IW	edium, high): ⁴² ode onding to the mode				
Cooling mode CFM Heating Design HVAC fan speed setting (e.g., low Cooling mode Heating Design total external static pressure (corwith the higher airflow above): 43 Room Name 1	w, medium, high): ⁴² ng mode responding to the mode	Coolir Desig Coolir Desig with the 1 2 3 4 5 6 7 8	ng mode CFM Heating m n HVAC fan speed setting (e.g., low, m ng mode Heating m n total external static pressure (corresp ne higher airflow above): 43 IW	edium, high): ⁴² ode onding to the mode				
Cooling mode CFM Heatin Design HVAC fan speed setting (e.g., low Cooling mode Heatin Design total external static pressure (corwith the higher airflow above): 43 Room Name 1	w, medium, high): ⁴² ng mode responding to the mode	Coolin Desig Coolin Desig With the Second Sec	ng mode CFM Heating m n HVAC fan speed setting (e.g., low, m ng mode Heating m n total external static pressure (corresp ne higher airflow above): 43 IW	edium, high): ⁴² ode onding to the mode				
Cooling mode CFM Heating Design HVAC fan speed setting (e.g., low Cooling mode Heating Design total external static pressure (corwith the higher airflow above): 43 Room Name 1	w, medium, high): 42 ng mode responding to the mode _ IWC Design Airflow (CFM)	Coolir Desig Coolir Desig with the 1 2 3 4 5 6 7 8	ng mode CFM Heating m n HVAC fan speed setting (e.g., low, m ng mode Heating m n total external static pressure (corresp ne higher airflow above): 43 IW	edium, high): ⁴² ode onding to the mode C Design Airflow (CFM)				